DEPARTMENT OF MECHANICAL ENGINEERING

TEAM 37 RACK CITY

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Problem Definition

Problem:

• Current sonar mounts for Autonomous Surface Vehicles (ASVs) are not remotely adjustable

Our Goal:

• Develop a mount system to improve floor imaging efficiency in Chesapeake Bay oyster deposit surveys

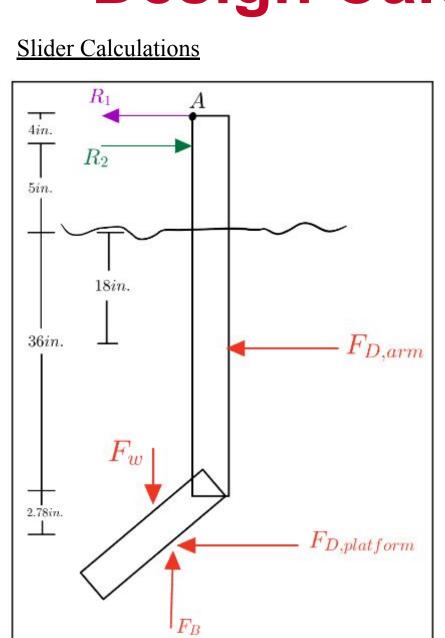
Desired Specifications:

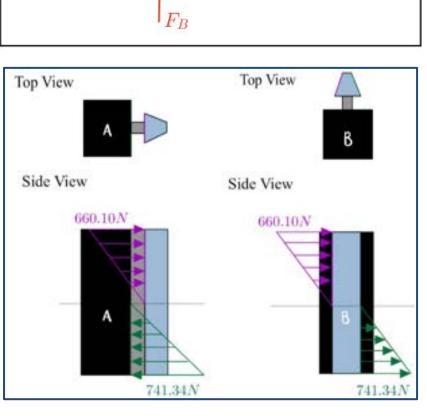
- Vertical translation from 0-3 feet underwater with 1 inch accuracy
- Angle adjustable from 16-45 degrees with 5 degree accuracy

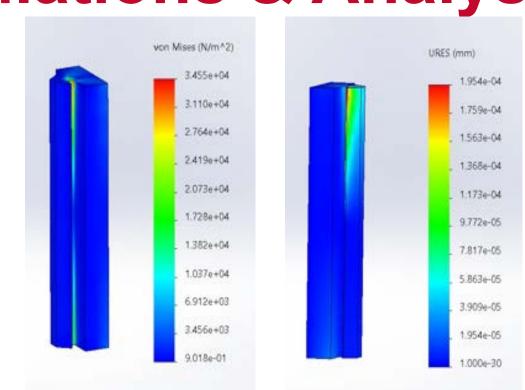




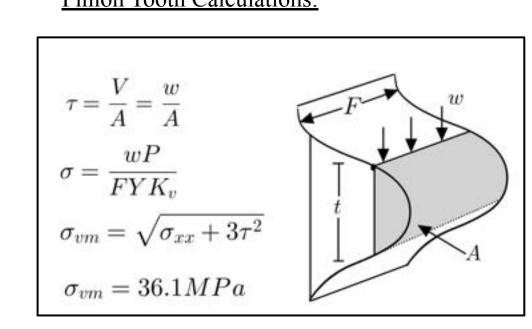
Design Calculations & Analysis







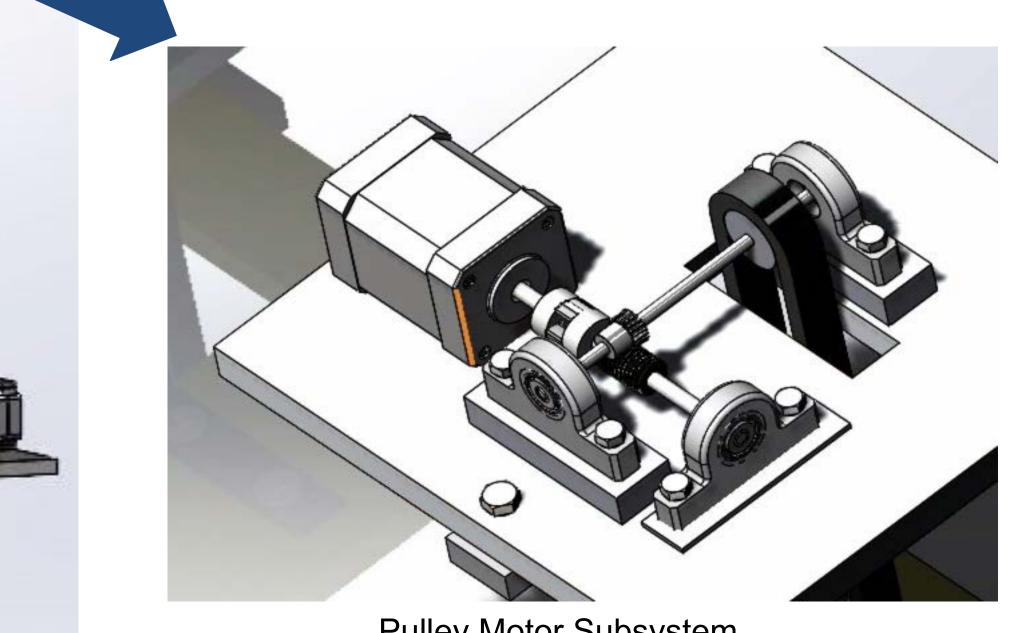
Pinion Tooth Calculations:



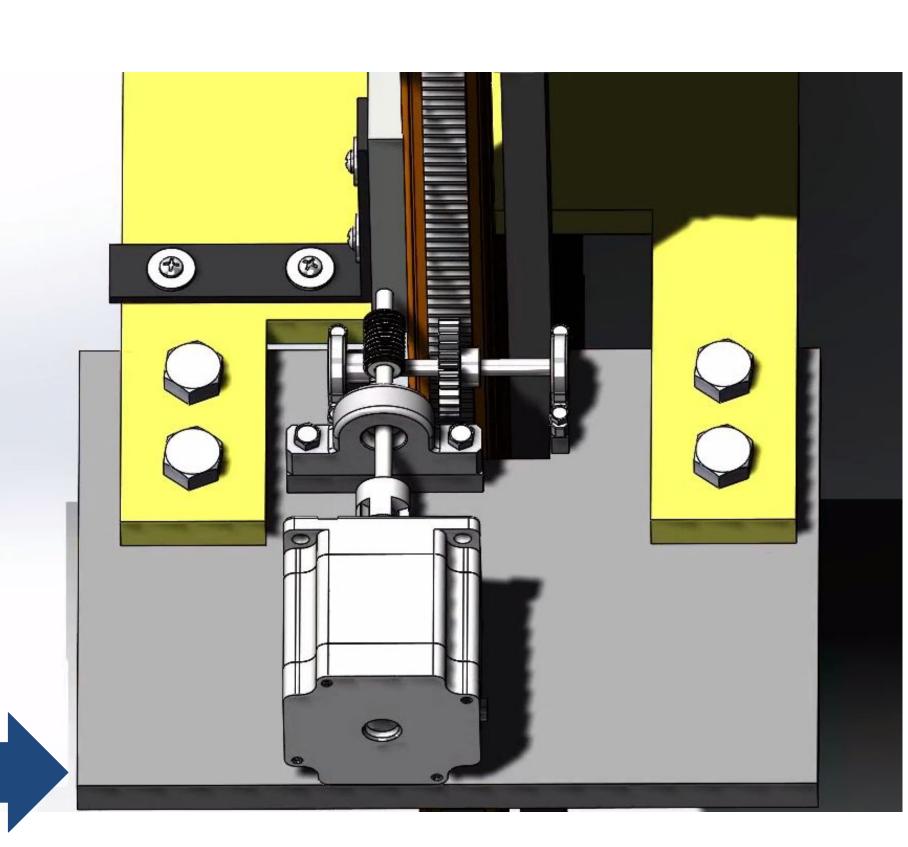
- % Stresses on pinion gear teeth 20 deg p angle
- z = 28; % # teeth p = (pi*g_d)/z; % (in) circular pitch
- P = pi/p; % (in) diametrical pitch
- F = 6*0.0393701; % (in) face width
- Y = 0.352; % Lewis form factor 20deg p angle
- $b_s = (W^*P)/(F^*Y); % (psi) bending stress in tooth$ b s metric = b s * 6894.76; % (Pa)

Stress Concentrations on Sliders Induced by Drag, $\sigma_{flange,top} = 39.527kPa$ Weight, and Buoyancy Forces: $\sigma_{flange,bottom} = 44.392kPa$ $\sigma_{head,top} = 56.419kPa$ $\sigma_{face,top} = 90.178kPa$ $\sigma_{head,bottom} = 63.370kPa$ $\sigma_{face,bottom} = 101.276kPa$

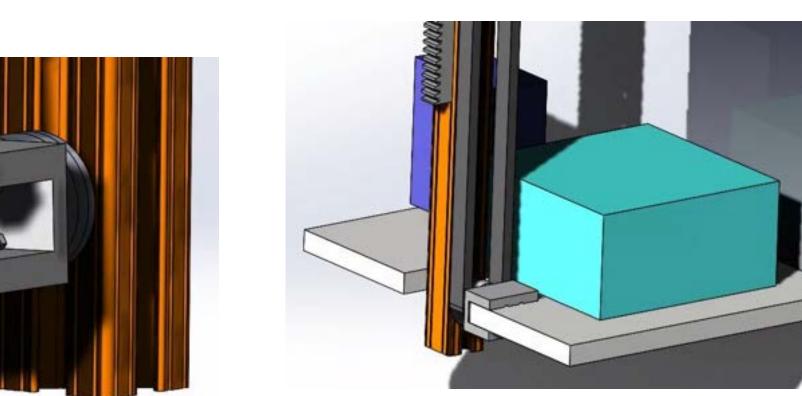
Final Design



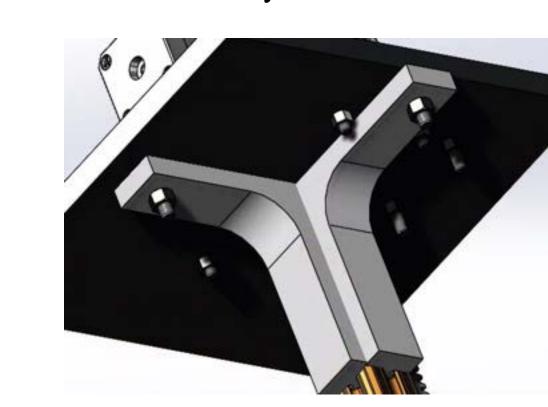
Pulley Motor Subsystem



Pinion Motor Subsystem

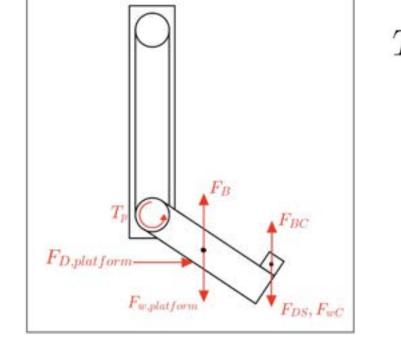


Bottom Platform Containing Sonar and Go-Pro



Custom T-slot Brackets

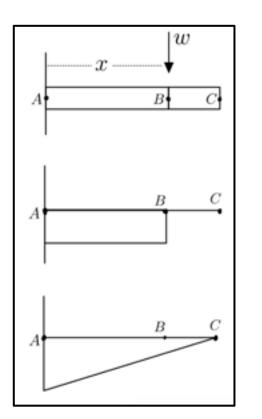
Pulley Motor Torque Calculations:

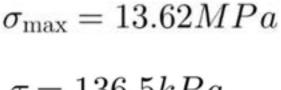


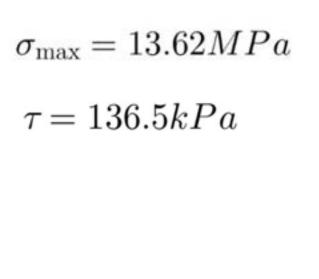
 $T_p = 4.42N \cdot m$

Bottom Pulley

Bottom Bolt Calculations:





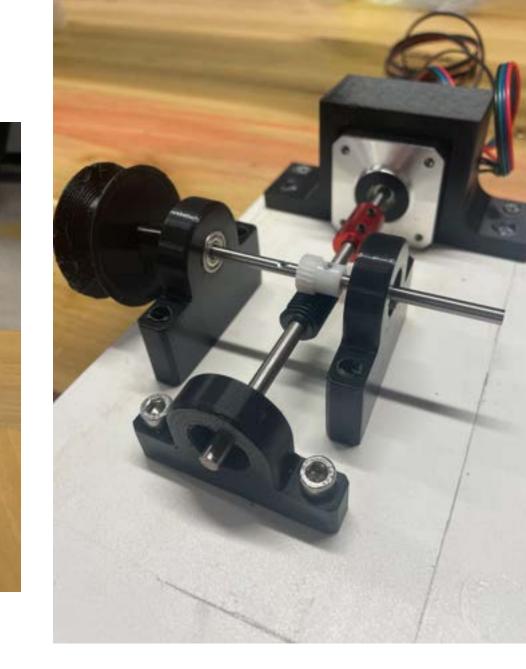


Prototype & Test Results



Bottom Pulley Flange

Prototype Pinion Motor Subsystem



Prototype Pulley Motor Subsystem

Slider Force Testing



Hanging scale test to determine if sliders can withstand calculated drag forces of ~36lbf